

EXPERIMENTAL INVESTIGATION OF CI ENGINE FUELED WITH DIESEL AND KEROSENE BLEND WITH COTTON SEED OIL

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ABSTRACT

Bio-fuel is renewable engine fuel that can be used directly in any existing, unmodified diesel engine. Bio-fuels create new markets for agricultural products and stimulate rural development because bio-fuels are generated from crops; they hold enormous potential for farmers. In the near future two-thirds of the people in the developing world will derive their income from agricultural products. In this study, the performance of a direct injection diesel engine has been investigated experimentally using 1st generation bio fuel (cottonseed oil) blends with fossil fuel like kerosene and diesel. The first generation bio-fuel (cottonseed oil) has been produced without transesterification reaction and we have made few blends samples of Cottonseed oil & Kerosene with Diesel fuel from literature review. First Flow analysis for various fuel blend which designed we have designed is carried out in this research work by using CFD software. Fuel Flow for various blends through injector of diesel engine is simulated and it gives the Density, velocity, pressure and volume distribution of fuel in combustion chamber. First analysis is done for pure diesel sample. Pure diesel results will be used to compare the results obtained in simulation of optimum fuel Blend. Optimum blend combinations are interpreted from the results obtained from CFD analysis then these optimum blends are tested on C.I. engine test rig. After that Comparison of flow characterizes of pure diesel with Blend sample will justify the technical feasibility of fuel blend which will conclude that the fuel blend sample can be used in single cylinder diesel engine without modification.

Keyword: - CFD, Fuel Blend, Flow Analysis, Cottonseed Oil, Kerosene.

1. INTRODUCTION

The India is presently confronted with the twin crises of fossil fuel depletion and environmental Degradation. Indiscriminate extraction and lavish consumption of fossil fuels have led to reduction in underground-based carbon resources. The search for alternative fuels, which promise a harmonious correlation with sustainable development, energy conservation efficiency and environmental preservation, has become highly pronounced in the present context. The fuels of bio-origin can provide a feasible solution to this worldwide petroleum crisis. By extending the research in the need of getting the optimum combination of fuel blend which we can use in direct injection diesel engine without going for the change in mechanical system of engine.

Many researchers have explored several alternative energy resources which have the ability to quench the ever increasing energy thirst. These resources are environment friendly. But the all resources are needed to be evaluated on the case to case basis for their Advantages, Disadvantages and their specific applications. Hence for we have decided to select a particular bio diesel fuel named 'COTTONSEED OIL' (CSO) and it will be blended with diesel in various different proportions. Performance of these different samples will be evaluated and tested in single cylinder CI engine. After evaluating the performance of Optimum fuel blend sample the complete flow analysis is being done for the blend sample. The results of blend sample are compared with samples of pure diesel and pure Cottonseed Oil.

1.1 NON-EDIBLE OIL AS AN ALTERNATIVE FUEL

The performance of I.C. engine using Karanja bio-diesel blended with diesel at various blending ratios has been evaluated. The test results indicated that the dual fuel combination of B40 can be used in diesel engine without making any engine modification [69]. Experimental investigation on waste frying oil and disclosed that the waste frying oil requires heating temp of 135°C to bring down the viscosity like diesel at 30°C. It was also observed that the performance was improved and carbon monoxide and smoke emissions were reduced using preheated waste frying oil and concluded that the waste frying oil preheated to 135° C could be used as a biodiesel for short term engine operation [71].

1.2 VEGETABLE OIL AND ITS BLENDS

Tests on some properties shows that viscosities were significantly higher and densities were marginally higher compared to diesel, vegetable oil has lower calorific values [8]. Both vegetable oils and alcohols such as Methanol, Ethanol are biomass derived renewable sources, but vegetable oils have properties more suitable to compression ignition engines compared to Alcohols. More than 30 different types of non edible oils are used in compression ignition engines. Blending of vegetable oils with some percentage of diesel fuel was a suitable method to reduce choking and for extended engine life [9].

2. METHODOLOGY

2.1 PROPERTIES OF TEST FUELS

Cottonseed oil can be easily mixed with diesel and kerosene in any proportion and can be used to partially substitute diesel. From the literature review the blend samples of cottonseed oil (CSO), Diesel (D) and Kerosene (Ke) are CSO10-D90, CSO20-D80, CSO25-D75, CSO30-D70, CSO35-D65, CSO40-Ke60, CSO10-Ke90, CSO20-Ke80, CSO25-Ke75, CSO30-Ke70, CSO35-Ke65, CSO40-D60 and D100 are being tested for properties like Kinematic Viscosity, Density, Flash Point, Cetane Number and Calorific Value are as follows:

Table -1: Properties of test fuels

Test Performed	Diesel	Cotton Seed Oil	Kerosene	CS O1 D90	CS O2 D80	CS O2 D75	CS O3 D70	CS O3 D65	CS O4 D60	CS O1 0-Ke90	CS O2 0-Ke80	CS O2 5-Ke75	CS O3 0-Ke70	CS O3 5-Ke65	CS O4 0-Ke60
Kinematic Viscosity (mm ² /s)	3.32	34.57	1.85	6.53	9.81	11.59	12.92	14.74	16.17	5.26	8.68	10.16	11.91	13.53	15.21
Density (kg/m ³)	823	934	783	836	847	856	861	867	883	801	817	827	832	841	852
Flash Point (°C)	56	198	43	73	86	92	101	106	114	59	77	85	92	102	108
Cetane Number	49.38	-	47.13	43.54	38.51	36.39	33.84	31.43	28.62	41.67	36.94	34.23	31.52	29.76	27.91
Calorific Value (KJ/Kg)	42843	39687	43386	42483	42119	41916	41782	41672	41506	42976	42513	42348	42105	42981	42843

2.2 FLOW ANALYSIS

Flow analysis of fuel flow from injector nozzle Geometry has prepared in ICEM CFD ANSYS Version 16. Considering Fuel injector nozzle layout 2D Geometry has prepared and discredited in ICEM CFD. Mesh file imported in Fluent Solver to simulate it for flow pattern with the help of pressure and velocity. The boundary conditions are the properties of blend samples i.e. Kinematic Viscosity (mm²/s), Density (Kg/m³), Flash Point (°C), Cetane Number & Calorific value (KJ/Kg) are used for simulation. The fuel blends from CFD analysis compared with pure diesel sample on the basis of spray cone angle are as follows:

Table -2: CFD analysis result for spray cone angle of test fuels

Samples	Diesel	CSO 10-D90	CSO 20-D80	CSO 25-D75	CSO 30-D70	CSO 35-D65	CSO 40-D60	CSO 10-Ke90	CS O20 - Ke80	CS O25 - Ke75	CSO 30-Ke70	CSO 35-Ke65	CSO 40-Ke60
Spray Cone Angle (deg)	8.4	7.9	7.1	6.7	5.1	4.6	4.1	8.7	8.1	7.8	7.5	6.2	5.3

It is found that an optimum blend with Cottonseed oil with Diesel is CSO25-D75 and for Cottonseed oil with Kerosene is CSO30-Ke70. These optimum blends are further tested in C.I engine for performance analysis to find best suitable blend which can be further use in C.I Engine

3. EXPERIMENTATION

The setup consists of single cylinder, four stroke, VCR (Variable Compression Ratio) Diesel engine connected to eddy current type dynamometer for loading as shown in figure 1. The engine specifications are given in Table 3. The compression ratio can be changed without stopping the engine and without altering the combustion chamber geometry by specially designed tilting cylinder block arrangement. Setup is provided with necessary instruments for combustion pressure and crank-angle measurements. These signals are interfaced to computer through engine indicator for Pθ-PV diagrams. Provision is also made for interfacing airflow, fuel flow, temperatures and load measurement. The set up has stand-alone panel box consisting of air box, two fuel tanks for duel fuel test, manometer, fuel measuring unit, transmitters for air and fuel flow measurements, process indicator and engine indicator, rotameters are provided for cooling water and calorimeter water flow measurement.

Table -3: Specification of engine

Manufacturer	Kirloskar Oil Engines Ltd., India
Model	TV1
Type	Vertical, Double cylinder, Water cooled, Four Stroke cycle, Compression Ignition Diesel Engine
Bore/stroke	87.5mm/110 mm
compression ratio	17.5 :1

Speed	1500 rpm
Eddy Current, Water Cooled	With Loading unit
Orifice	Diameter of orifice = 0.02m Coefficient of discharge of Orifice $C_d = 0.62$

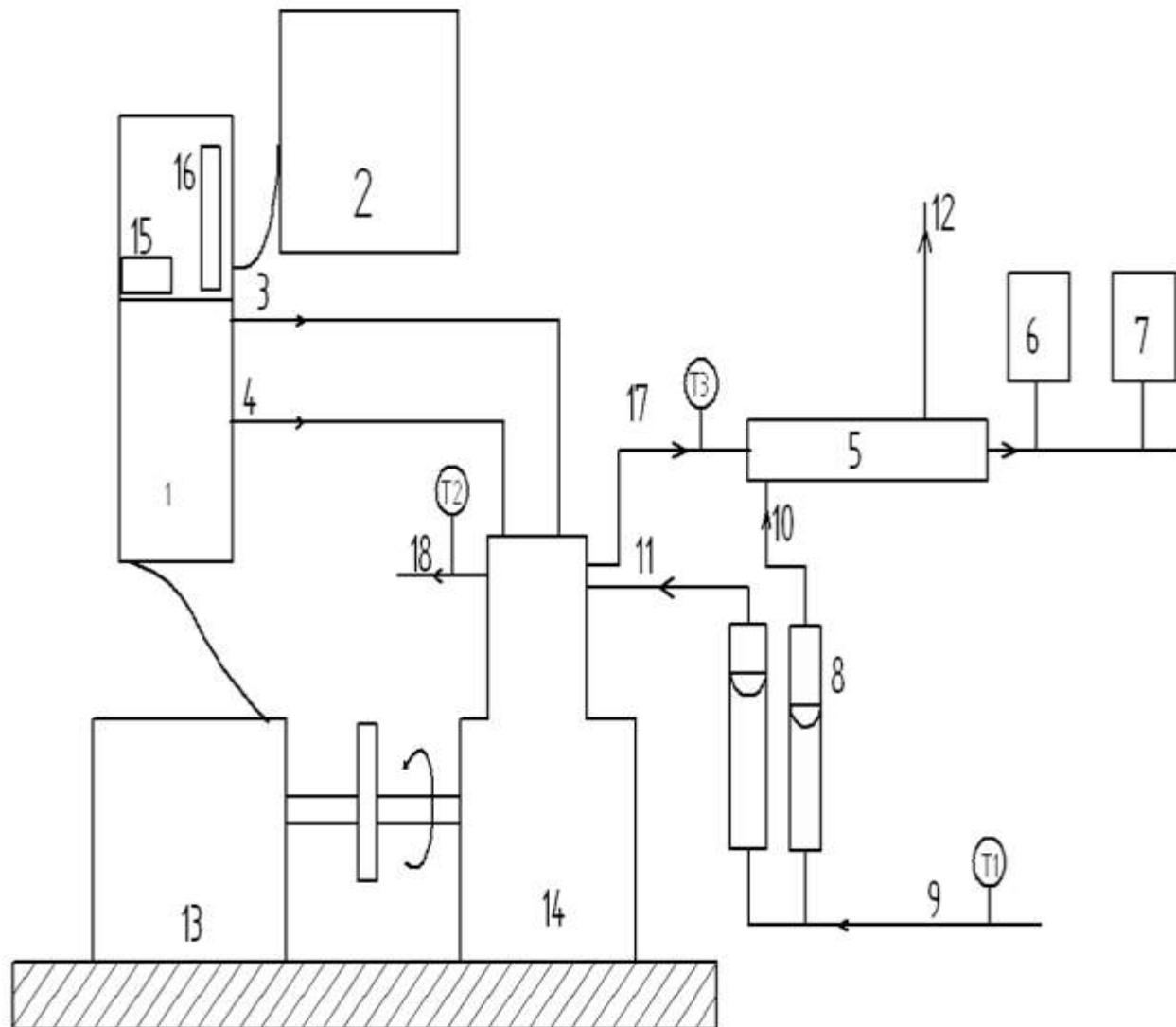


Fig -1: Schematic diagram of experimental setup

1 = Control Panel, 2 = Computer system, 3 = Diesel flow line, 4 = Air flow line, 5 = Calorimeter, 6 = Exhaust gas analyzer, 7 = Smoke meter, 8 = Rotameter, 9 = Calorimeter inlet water temperature, 10 = Calorimeter outlet water temperature, 11 = Dynamometer, 12 = CI Engine, 13 = Speed measurement, 14 = Burette for fuel measurement, 15 = Exhaust gas outlet, 16 = Outlet water temperature, T1= Inlet water temperature, T2 = Outlet water temperature, T3 = Exhaust gas temperature.

5. RESULTS AND DISCUSSIONS

5.1 FLOW ANALYSIS RESULTS

The flow analysis of fuel spray through the fuel injector is analyzed. As the viscosity of new blend samples are varied as compared to the pure Diesel sample. Velocity is the function of viscosity. Maximum Flow velocity occurred at bends in all blend samples. To predict the effect of viscosity on the fluid flow velocity distribution is obtained for D100, CSO25-D75 and CSO30-KE70. Penetration velocity of pure diesel sample i.e.3680 m/s is nearer to CSO30-KE70 sample i.e.4250m/s.

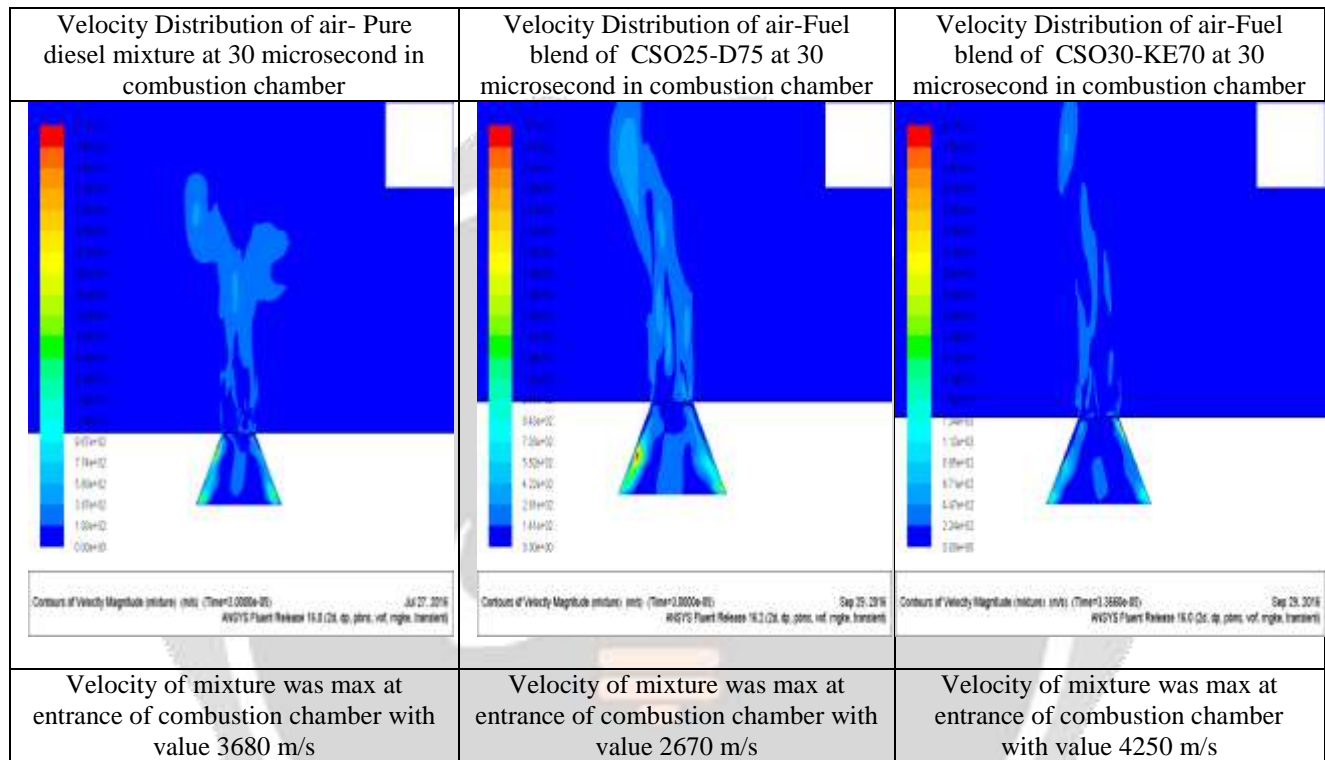


Fig -2: Comparison of Velocity Distribution Profiles for fuel blends inside combustion chamber.

5.2 EXPERIMENTAL RESULT

Different experimental results like brake specific fuel consumption, brake thermal efficiency, etc are compared with load as shown below:

5.2.1 Brake thermal efficiency

The brake thermal efficiency plots in chart 1 show an increase of brake thermal efficiency with increase in the engine load as the amount of diesel in the blend increases. Maximum brake thermal efficiency of CSO30-KE70 is 41.53%, CSO25-D75 is 36.01% and for pure diesel is 43.16%. The comparison of brake thermal efficiency of Cottonseed oil-diesel and Cottonseed oil -kerosene blend with diesel which indicates that brake thermal efficiency increases with increasing load in all cases. CSO30-KE70 blend gives result slightly more than the pure diesel.

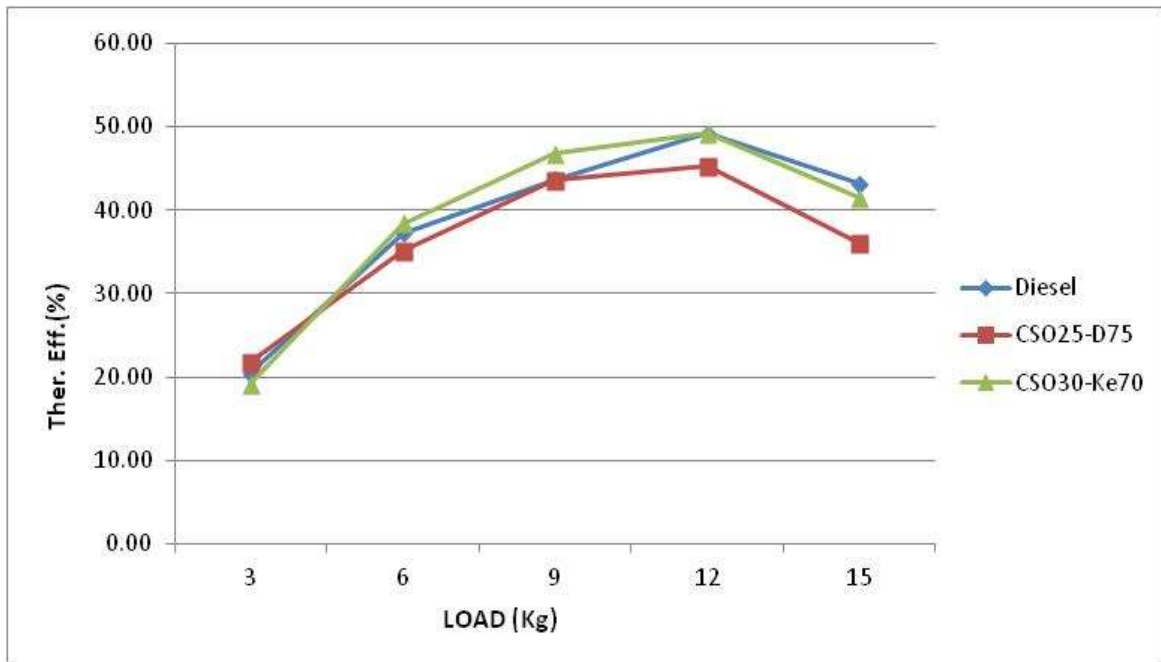


Chart -1: Comparison of Brake Thermal Eff. of CSO-Diesel-Kerosene blends with load

5.2.2 Brake specific fuel consumption (BSFC)

The brake specific fuel consumption for pure diesel is 0.19 (Kg/KW-Hr), CSO30-KE70 is 0.21(Kg/KW-Hr) and CSO25-D75 is 0.24 (Kg/KW-Hr) From chart 2 it shows the comparison of brake specific fuel consumption (kg/KW-hr) of Cottonseed oil-diesel and Cottonseed oil -kerosene blends with load. Chart indicates that BSFC reduces with increase in load in all cases. This may due to higher viscosity and lower calorific value. CSO30-KE70 blend gives result much equivalent to pure diesel.

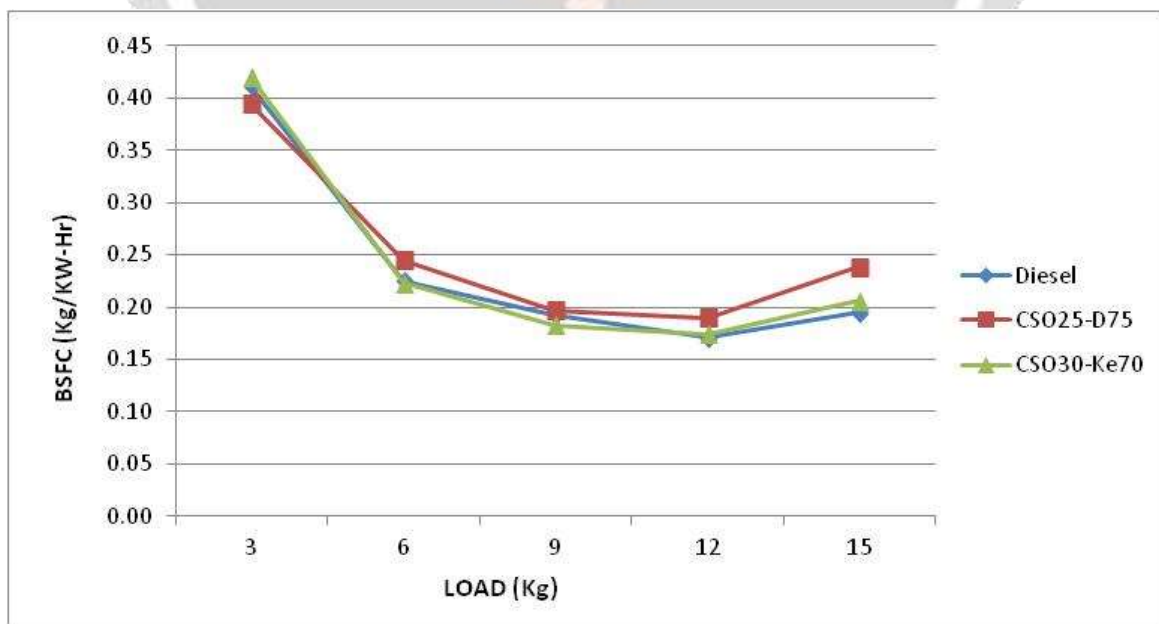


Chart -2: Comparison of BSFC of CSO-Diesel-Kerosene blends with load

4. CONCLUSIONS

The CFD analysis and graphical results show that diesel has better performance characteristics than biodiesel and biodiesel blends. The best performance as good alternative fuel and evaluated the good results to prove the better performance of cottonseed oil and kerosene and compared with diesel then the conclusion are review from them.

1. The spray cone angle for the blends CSO-DI (25-75) i.e. 6.7 deg and CSO-KE (30-70) i.e. 7.5 deg is lesser than pure diesel i.e. 8.4 deg caused by higher density of biodiesel blends than base diesel. The possible reason is that spray cone angle is a main function of charge density, which directly relates with the in-cylinder pressure.
2. Poor air entrainment was caused by biodiesel having higher viscosity. Air entrainment increases because of high penetration distance. Air entrainment is more in case of biodiesel blends than base diesel as penetration distance is more for CSO-KE (30-70).
3. Volume fraction distribution profiles for CSO-KE (30-70) i.e. 0.3% inside the cylinder with pure diesel i.e. 0.3% from which it is found that Profiles of Cottonseed oil 30% with Kerosene 70% fuel blend profiles matched to greatest extent.
4. Technical Feasibility like BSFC and Break Thermal Efficiency for CSO-KE (30-70) i.e. 0.21 (Kg/KW-Hr) and 41.53% for CSO-DI (25-75) i.e. 0.24 (Kg/KW-Hr) and 36.01% compared with pure diesel i.e. 0.19 (Kg/KW-Hr) and 43.16% respectively, hence Cottonseed oil 30% with Kerosene 70% blend is validated and found matching with diesel fuel.
5. Spray characteristics for Fuel blend are calculated and compare it with pure diesel sample, Also Density, Velocity, Volume fraction & Injection pressure distribution profiles created in CFD fluent for Fuel Blend samples Comparing the profiles conclude that **Cottonseed Oil 30% - Kerosene 70%** is most similar blend to pure diesel.
6. Behavior of blend properties for sample of diesel, cottonseed oil & kerosene blend inside combustion chamber is studied.

From the flow analysis of different samples an optimum blend found from the CFD analysis on the basis of pressure, density, volume and velocity is selected for the experimentation on C.I. engine.

The results for BSFC and Break Thermal Efficiency are found from the experimentation is nearly matched with the CFD analysis

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